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FOR  
**Method and System for Controlling Hourglass Deformations of Solid  
Elements in Finite Element Analysis**

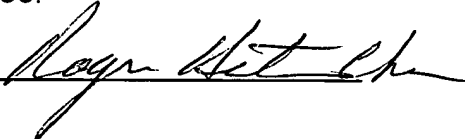
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# **Method and System for Controlling Hourglass Deformations of Solid Elements in Finite Element Analysis**

## **BACKGROUND OF THE INVENTION**

### **1. Field of the Invention**

**[0001]** The present invention generally relates to method, system and software product used in structural design and analysis using finite element analysis, more particularly to control zero-energy hourglass mode deformations of solid elements in finite element analysis.

### **2. Description of the Related Art**

**[0002]** Finite element analysis (FEA) is a computerized method widely used in industry to model and solve engineering problems relating to complex systems. FEA derives its name from the manner in which the geometry of the object under consideration is specified. With the advent of the modern digital computer, FEA has been implemented as FEA software. FEA software can be classified into two general types, implicit FEA software and explicit FEA software. Implicit FEA software uses an implicit equation solver to solve a system of coupled linear equations. Such software is generally used to simulate static or quasi-static problems. Explicit FEA software does not solve coupled equations but explicitly solves for each unknown assuming them uncoupled. Explicit FEA software usually uses central difference time integration which requires very small solution cycles or time steps for the method to be stable and accurate. Explicit FEA software is generally used to simulate short duration events where dynamics are important such as impact type events.

**[0003]** The user of FEA software creates a model of the system to be analyzed using elements. An element represents a finite region of the system. Within each element, the unknown quantity is assumed to take a simple form within the domain of the element. For explicit FEA software, the unknown

quantity is usually acceleration. For implicit FEA software, the unknown quantity may be displacement, velocity, temperature, or others.

**[0004]** By assuming a simple form of the unknown within an element, and by using many elements, complex behaviors can be simulated within a reasonable time frame with the FEA software. One simple example is a linear spring element with the displacement at the ends of the spring as the unknown quantity. The displacement field is assumed to vary linearly along the length of the spring. Therefore, if we solve for the displacement at the ends of the spring, we can easily evaluate the displacement at any point along the spring. The compatible strain and stress fields for a linear spring element are constant over the element length and are easily evaluated from the end displacements and material properties.

**[0005]** The points on the element where the unknown is solved are called nodes. The linear spring has a node at each end. If we place a third node at the middle of the spring element, the displacement field could then be assumed to vary as a quadratic function and the compatible strain and stress fields would be linear. The common element types are solid elements for modeling volumes, shell elements for modeling thin parts dominated by bending, beam elements for modeling beams, and spring or truss elements for modeling springs and trusses. Each element is assigned a material type and appropriate material properties. By choosing appropriate material types and properties, metals, plastics, foams, soil, concrete, rubber, glass, fluids and many other materials can be modeled. The user must also specify the boundary conditions, loads, and initial conditions to complete the model. To accurately simulate complex system behavior, many elements are needed. Today, typical FEA models of entire automobile are made of more than 500,000 elements.

**[0006]** Once the model is defined, FEA software can perform a simulation of the physical behavior under the specified loading or initial conditions. FEA software is used extensively in the automotive industry to simulate front and side impacts of automobiles, occupant dummies interacting with airbags, and the forming of body parts from sheet metal. Such simulations provide valuable

insight to engineers who are able to improve the safety of automobiles and to bring new models to the market more quickly.

**[0007]** Solid elements are typically used for modeling thick parts or solid bodies. In three dimensions, a solid element can be shaped like brick or hexahedron. The lowest order brick element has a node at each corner and is thus called the 8-node brick or hexahedral element. The 8-node brick element can be assumed to have a displacement (or other unknown) field that varies linearly along the edges between the nodes. Throughout the element domain, the displacement field has linear terms and cross terms but no quadratics or higher order terms. The compatible stress and strain fields have linear terms within the element domain. There are other types of solid elements such the 6-node pentahedral element.

**[0008]** To calculate the nodal forces that are generated by the stress within an element, the calculated stress tensor is multiplied by terms that account for the element's geometry and then integrated over the domain of the element. For many materials, the stress field does not have a linear relationship to the strain field, so closed form integration is not possible. Instead, numerical integration such as Gauss-Legendre quadrature is routinely used. Numerical integration of an 8-node brick element can be done by defining two Gauss-Legendre integration points in each spatial direction for a total of 8 integration points. Such element is said to have full integration or rank sufficient integration. Full integration guarantees that all possible modes of deformation generate stress in the element. Alternatively, numerical integration of an 8-node brick element can be done with a single Gauss-Legendre integration point. Such an element is called an under-integrated or rank deficient element.

**[0009]** The under-integrated element is faster than the fully integrated element because only one stress evaluation is required per element per solution cycle rather than eight in a fully integrated element. This speed advantage is very noticeable in an explicit FEA solution where the stress evaluation dominates the solution time. An under-integrated element is also more robust in large deformation calculations because the terms evaluated at the integration point

remain well conditioned at larger deformation. This follows directly from the location of the single point, at the element center, rather than shifted towards the corners as with the fully integrated element.

**[0010]** The disadvantage to numerical integration with only one integration point is that some modes of deformation do not contribute to the strain or stress field at the integration point. These modes which were originally noticed in finite difference calculations in two dimensions in the 1960's are historically called hourglass modes because of their shape as shown in FIGS. 1A and 1B. These modes are also called zero-energy modes because they do not generate any strain energy in the element. FIG. 2A shows an 8-node brick element with one integration point in the center of the element. The 8-node element has four zero-energy modes in each of three spatial directions for a total of 12. The four modes in one of the spatial directions are shown in FIG. 2B.

**[0011]** To use under-integrated elements in a finite element analysis, it is necessary to limit the deformation due to the zero-energy modes. Otherwise, the deformation due to these zero-energy modes may grow too large and dominate the solution. Some existing methods for controlling hourglass deformation are found in the following papers: "A Uniform Strain Hexahedron and Quadrilateral with Orthogonal Hourglass Control" by Flanagan and Belytschko (1981); "Efficient Implementation of Quadrilaterals with High Coarse-Mesh Accuracy" by Belytschko and Bachrach (1985); and "Assumed Strain Stabilization of the Eight Node Hexahedral Element" by Belytschko and Bindeman (1993).

**[0012]** To be accurate for large deformation analysis, these existing hourglass control methods calculate an increment of hourglass force using the current element geometry to scale the force increments. The force increments are summed over time to obtain the total hourglass force. Because the element geometry changes during the solution, the increments of hourglass force calculations are based on a variable stiffness. This nonlinearity in the stiffness can cause residual hourglass deformation to remain in an element after some load is applied to the element and then removed. One exemplary application

that demonstrates residual hourglass deformation is an automobile tire model that was used to study its behavior during skidding. The tire model was first pressed onto the road with a force to represent the weight of the vehicle. The tire model was then moved with a prescribed motion to start the tire rolling. After some time, the prescribed motion caused the rolling tire to skid. 8-node brick elements were used to model the tire tread. If we observe one element that models a small part of the tire tread, we would see it rotating with the tire. At some point, the element would contact the road and be compressed by the weight of the automobile. Because the tire is skidding, we would also see frictional forces cause shear deformation of the element. After a few moments, the rotation of the tire would cause the deformed element to separate from the road surface. With the compressive and shear loading removed, we would expect the element to spring back to its undeformed geometry. However, after a few rotations of the tire, we observed the brick elements of the tread failing to spring back to their undeformed geometries.

**[0013]** It is therefore desirable to have a new method to control hourglass deformations for solid elements in the applications that the traditional methods do not work well.

## SUMMARY OF THE INVENTION

**[0014]** This section is for the purpose of summarizing some aspects of the present invention and to briefly introduce some preferred embodiments. Simplifications or omissions may be made to avoid obscuring the purpose of the section. Such simplifications or omissions are not intended to limit the scope of the present invention.

**[0015]** According to one aspect of the present invention, the calculation of a set of nodal forces to resist hourglass deformation of solid elements can be summarized by the following exemplary procedure for a single 8-node solid brick element during a particular solution cycle: 1) Establish the initial element coordinate system using the global coordinates of the nodes of the initially undeformed element; 2) Establish the current element coordinate system using the global coordinates of the nodes of the current deformed element; 3)

Calculate initial nodal coordinates by transforming the coordinates of the nodes of the initially undeformed element from the global coordinate system to the initial element coordinate system; 4) Calculate current nodal coordinates by transforming the coordinates of the nodes of current deformed element from the global coordinate system to the current element coordinate system; 5) Evaluate the hourglass shape vectors using the initial nodal coordinates, the hourglass base vectors, and the derivatives of the tri-linear solid element shape functions which are also based on the initial nodal coordinates; 6) Calculate the magnitude of hourglass deformation of all hourglass modes using hourglass shape vectors, the initial nodal coordinates, and current nodal coordinates; 7) Evaluate generalized hourglass forces using the magnitude of the hourglass deformation, the initial nodal coordinates, and material constants such as Poisson's ratio and shear modulus of the element; 8) Calculate a set of nodal forces to resist hourglass deformation using the generalized hourglass forces and the hourglass shape vectors; 9) Transform the set of nodal forces from the current element coordinate system to the global coordinate system and add to the global force array at corresponding nodes.

**[0016]** According to another aspect of the present invention, the calculation of an element stabilization matrix added to the rank-deficient element stiffness matrix such that sum of the matrices is rank-sufficient can be summarized by the following exemplary procedure for a single 8-node solid brick element during a particular solution cycle: 1) Establish the initial element coordinate system using the global coordinates of the nodes of the initially undeformed element; 2) Establish the current element coordinate system using the global coordinates of the nodes of the current deformed element; 3) Calculate initial nodal coordinates by transforming the coordinates of the nodes of the initially undeformed element from the global coordinate system to the initial element coordinate system; 4) Calculate current nodal coordinates by transforming the coordinates of the nodes of current deformed element from the global coordinate system to the current element coordinate system; 5) Evaluate the hourglass shape vectors using the initial nodal coordinates, the hourglass base

vectors, and the derivatives of the tri-linear solid element shape functions which are also based on the initial nodal coordinates; 6) Calculate all terms of the element stabilization matrix using the hourglass shape vectors, initial nodal coordinates, and elastic material constants for the element, such as Poisson's ratio and shear modulus. 7) Transform the element stabilization matrix from the initial element coordinate system to the global coordinate system before adding it to the global stiffness matrix.

**[0017]** According to yet another aspect of the present invention, the set of nodal forces to resist hourglass deformation is designed to return to zero when the hourglass deformation returns to zero. In the exemplary tire simulation, this allows the tire to spring back to its initial geometry for any number of tire rotations. The present invention calculates the hourglass deformation by directly comparing the current geometry with the initial geometry. With this method, error cannot accumulate over time and elements can always spring back to their initial shape.

**[0018]** According to yet another aspect of the present invention, an embedded coordinate system is defined to transform and rotate with the element. This allows any amount of rigid body translation and rotation of the element to occur without affecting the hourglass deformation measure.

**[0019]** Other objects, features, and advantages of the present invention will become apparent upon examining the following detailed description of an embodiment thereof, taken in conjunction with the attached drawings.

#### BRIEF DESCRIPTION OF THE DRAWINGS

**[0020]** These and other features, aspects, and advantages of the present invention will be better understood with regard to the following description, appended claims, and accompanying drawings as follows:

**[0021]** FIGS. 1A and 1B illustrate an exemplary hourglass mode of a two-dimensional finite element.

**[0022]** FIGS. 2A and 2B show four exemplary zero-energy modes of a three-dimensional 8-node brick element.



**[0023]** FIG. 3 depicts the relationship between the global coordinate system and an element coordinate system.

**[0024]** FIG. 4A is the flow chart of calculating nodal forces to counter the hourglass deformation of a solid element in the present invention.

**[0025]** FIG. 4B is the flow chart of calculating element stabilization matrix of a solid element in the present invention.

**[0026]** FIGS. 5A to 5E are the formulas and tables for calculating the nodal hourglass forces of an exemplary three-dimensional 8-node brick element.

**[0027]** FIGS. 5F to 5H are the formulas for evaluating the stabilization matrix for an exemplary 8-node brick element.

**[0028]** FIG. 6 depicts a block diagram of an exemplary computer, in which the present invention may be implemented.

## **DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS**

**[0029]** In the following description, numerous specific details are set forth in order to provide a thorough understanding of the present invention. However, it will become obvious to those skilled in the art that the present invention may be practiced without these specific details. The descriptions and representations herein are the common means used by those experienced or skilled in the art to most effectively convey the substance of their work to others skilled in the art. In other instances, well-known methods, procedures, components, and circuitry have not been described in detail to avoid unnecessarily obscuring aspects of the present invention.

**[0030]** Reference herein to “one embodiment” or “an embodiment” means that a particular feature, structure, or characteristic described in connection with the embodiment can be included in at least one embodiment of the invention. The appearances of the phrase “in one embodiment” in various places in the specification are not necessarily all referring to the same embodiment, nor are separate or alternative embodiments mutually exclusive of other embodiments. Further, the order of blocks in process flowcharts or diagrams representing one or more embodiments of the invention do not inherently indicate any particular order nor imply any limitations in the invention.

**[0031]** To facilitate the description of the present invention, it deems necessary to provide definitions for some terms that will be used throughout the disclosure herein. It should be noted that the definitions following are to facilitate the understanding and describe the present invention according to an embodiment. The definitions may appear to include some limitations with respect to the embodiment, the actual meaning of the terms has applicability well beyond such embodiment, which can be appreciated by those skilled in the art:

**[0032]** FEA stands for Finite Element Analysis.

**[0033]** Implicit FEA refers to the use of an implicit solver to solve coupled linear equations for the unknowns. Frequently, this takes the form  $Kx = F$ ,

where  $K$  is the global stiffness matrix,  $x$  is the unknown displacement array and  $F$  is the global force array.

**[0034]** Explicit FEA refers to the way the acceleration values of each node are solved directly or explicitly for each degree of freedom. This takes the form  $M a = F$ , where  $M$  is the mass array,  $a$  is the acceleration array and  $F$  is the global force array. The equations are uncoupled by assuming that the mass array is non-zero terms only on the diagonal.

**[0035]** Time domain analysis refers to a FEA that simulates a physical phenomenon in time domain.

**[0036]** Dynamic analysis refers to a FEA that accounts for the mass and inertia effects of a structure. In general, there are two classes of dynamic analysis: time domain and frequency domain.

**[0037]** Solution cycle and cycle are used interchangeably to refer to the numbered solution states starting with cycle 0 at time 0.

**[0038]** The time step refers to an interval of time between subsequent cycles.

**[0039]** Two-dimensional solid element refers to 4-node 2D continuum element, such as plane strain and axisymmetric element.

**[0040]** Three-dimensional solid element refers to finite element such as 8-node hexahedral element, 6-node pentahedral element.

**[0041]** Global coordinate system refers to the coordinate system used to define the FEA model.

**[0042]** Initial element coordinate system refers to a local coordinate system affixed to the element based on the element's initial undeformed geometry at cycle 0.

**[0043]** Current element coordinate system refers to a local coordinate system affixed to the element based on the element's current deformed geometry at current cycle.

**[0044]** Initial nodal coordinates refer to coordinates of the node for the undeformed geometry at cycle 0.

**[0045]** Current nodal coordinates refer to the nodal coordinates for the deformed geometry at the current cycle.

**[0046]** Integration point refers to the particular location within a finite element that is used for numerical integration during the element force evaluation.

**[0047]** Mode shape refers to fundamental vibration mode shapes in a dynamic analysis.

**[0048]** Zero-energy modes and Hourglass modes are interchangeable in this specification.

**[0049]** Embodiments of the present invention are discussed herein with reference to FIGS. 1-6. However, those skilled in the art will readily appreciate that the detailed description given herein with respect to these figures is for explanatory purposes as the invention extends beyond these limited embodiments.

**[0050]** Referring now to the drawings, FIG. 1A shows a two-dimensional (2D) finite element **110** with one integration point **115** at the center of the element. FIG. 1B shows a plurality of 2D finite elements **120** demonstrating the zero-energy mode of deformation. The shape formed by two adjacent elements resembles an hourglass, which motivated the naming of hourglass modes.

**[0051]** FIG. 2A shows an exemplary three-dimensional (3D) 8-node brick element **260** with one integration point **265** in the center of the element. FIG 2B shows the four zero-energy modes **282**, **284**, **286** and **288** in one of the spatial directions for the 3D 8-node brick element. These modes are frequently called hourglass modes although not all the zero-energy modes of the 3D 8-node brick element resemble an hourglass.

**[0052]** Referring now to FIG. 3, two coordinate systems used in the present invention are shown. The global coordinate system **310** is used for defining the overall FEA model geometry. The element coordinate system **320** is embedded in the element. Both of these coordinate systems are Cartesian coordinate systems having three orthogonal axes representing three spatial directions for the three dimensional space. The element coordinate system is a local coordinate system that is defined for each solid element in terms of the nodal coordinates. The initial element coordinate system is defined as the element coordinate system calculated using the initial coordinates of the nodes at the

cycle 0. During a simulation, the locations of the nodes are updated each cycle if the element becomes deformed and undergoes translation and rotation. The element coordinate system is recalculated at each solution cycle so that it translates and rotates with the element. At a particular cycle, the element coordinate system defined by the nodal coordinates during that cycle is referred as the current element coordinate system.

**[0053]** In one embodiment, calculating the element coordinate system of an 8-node brick element uses the following procedure: 1) a trial element coordinate system is first aligned with the axes connecting the centers of opposite faces of the 8-node brick element; 2) an iterative scheme is then applied to impose orthogonality on the element coordinate system. In each iteration, a new trial coordinate system is defined as the average direction of the previous trial coordinate system and the cross product of the other two previous trial coordinate system axes. When the dot product of each new trial system axis with the previous trial system axis is nearly one, the iterations are stopped and the element coordinate system is assumed to be the final trial system.

**[0054]** In another embodiment, by assuming that a 6-node pentahedral element is actually an 8-node hexahedral element with a collapsed face and two pairs of coincident nodes, the element coordinate system of a 6-node pentahedral element is formed in exactly the same way as described for the 8-node hexahedral element. The transformation between the global coordinate system and the element coordinate system comprises a rotational and a translational operation.

**[0055]** Referring now to FIG. 4A, which shows the flow chart **400** for calculating nodal forces to counter hourglass deformation for one solid element during one solution cycle in the present invention. At **402**, the initial element coordinate system is established using the global coordinates of the nodes of the initially undeformed element. At **404**, the current element coordinate system is established using the global coordinates of the nodes of the currently deformed element. At **406**, the initial nodal coordinates are calculated by transforming the coordinates of the nodes of the initially undeformed element

from the global coordinate system to the initial element coordinate system. At **408**, the current nodal coordinates are calculated by transforming the coordinates of the nodes of the currently deformed element from the global coordinate system to the current element coordinate system. At **410**, the hourglass shape vectors are evaluated using the initial nodal coordinates, the derivatives of the tri-linear solid element shape functions which are also evaluated using the initial nodal coordinates, and the hourglass base vectors. At **412**, the magnitude of the all modes of hourglass deformation is calculated using hourglass shape vectors, the initial nodal coordinates, and the current nodal coordinates. At **414**, the generalized hourglass forces are evaluated using the magnitudes of all modes of hourglass deformation, the initial nodal coordinates, and elastic material constants of the element, such as Poisson's ratio and shear modulus.

**[0056]** In one embodiment, there are 12 modes of hourglass deformation in an 8-node brick element. At **416**, a set of nodal forces are calculated to resist hourglass deformation using the generalized hourglass forces and the hourglass shape vectors. At **418**, the set of nodal forces are transformed from the current element coordinate system to the global coordinate system and added to the global force array at the corresponding nodes.

**[0057]** FIG. 4B shows the flow chart **450** for calculating an element stabilization matrix that is added to the rank-deficient element stiffness matrix such that sum of the matrices is rank-sufficient. At **452**, the initial element coordinate system is established using the global coordinates of the nodes of the initially undeformed element. At **454**, the current element coordinate system is established using the global coordinates of the nodes of the currently deformed element. At **456**, the initial nodal coordinates are calculated by transforming the coordinates of the nodes of the initially undeformed element from the global coordinate system to the initial element coordinate system. At **458**, the current nodal coordinates are calculated by transforming the coordinates of the nodes of the currently deformed element from the global coordinate system to the current element coordinate system. At **460**, the

hourglass shape vectors are evaluated using the initial nodal coordinates, the derivatives of the tri-linear solid element shape functions which are also evaluated using the initial nodal coordinates, and the hourglass base vectors. At **462**, the terms of the element stabilization matrix are calculated using the hourglass shape vectors, initial nodal coordinates, and elastic material constants for the element, such as Poisson's ratio and shear modulus. At **464**, the element stabilization matrix is transformed from the initial element coordinate system to the global coordinate system. At **466**, the element stabilization matrix terms are added into the global stiffness matrix. For implicit FEA, the nodal forces for resisting hourglass deformation and the stabilization matrix of the solid element are added to the global force array and global stiffness matrix, respectively. In the case of explicit FEA, only the nodal forces for resisting hourglass deformation is added to the global force array.

**[0058]** The formulas and tables listed in FIGS. 5A-5D are used for controlling hourglass deformations of an exemplary 8-node brick element of the present invention. Formula **510** in FIG. 5A is used for calculating the hourglass shape vectors  $\bar{\gamma}_{\alpha i}$  for the 8-node brick element.  $\Gamma_{\alpha i}$  is a term from one of the four hourglass base vectors as listed in table **580** in FIG. 5C,  $\bar{x}_J$  is the initial nodal coordinate of node J, and  $\bar{B}_{ij}$  is the spatial derivative of the tri-linear shape functions with respect to the initial coordinate directions. The initial nodal coordinates have been transformed to the local element coordinate system. Similarly,  $\bar{B}_{ij}$  and  $\bar{\gamma}_{\alpha i}$  are evaluated using the initial nodal coordinates transformed to the local element coordinate system. Formula **520** in FIG. 5A calculates the magnitude of the hourglass deformation  $\hat{g}_{\alpha i}$  for hourglass mode  $\alpha$  in spatial direction  $i$ . The current nodal coordinates of node J,  $\hat{x}_J$  have been transformed to the current element coordinate system. Formulas **530** in FIG. 5B are used to evaluate the generalized forces  $\bar{G}_{\alpha}$ ,  $\bar{G}_{ij}$  and  $\bar{G}_{i4}$ . In **530**,  $\mu$  is the shear modulus and  $\nu$  is the Poisson's ratio of the material model specified for the element. Formulas **535** in FIG. 5B are used to evaluate the  $\bar{H}_{\alpha}$  and  $\bar{H}_{ij}$  terms in Formulas **530**. The  $\bar{H}_{\alpha}$  and  $\bar{H}_{ij}$  terms are evaluated from the initial nodal coordinates

transformed to the initial element coordinates system  $\bar{x}_u$  and the referential coordinates  $\xi_u$  listed in table **585** as shown in FIG. 5D. In Formulas **530** and **535**, the subscripts i, j, and k permute based on the values listed in table **590** as shown in FIG. 5E. Formula **540** in FIG. 5B evaluates the nodal hourglass forces that resist hourglass deformation from the generalized forces and the hourglass shape vectors.

**[0059]** FIGS. 5F-5H are the formulas used to calculate the element stabilization matrix for an exemplary 8-node brick element. In the embodiment for the 8-node brick element, formulas **592** in FIG. 5F define the 8-element array  $\gamma_j$  and the 8-element transpose array  $\gamma_j^T$ . The components of these arrays are  $\bar{\gamma}_{aj}$  calculated according to Formula **510** in FIG. 5A. Formulas **594** in FIG. 5G define the three diagonal blocks of the element stabilization matrix. Each block is an 8x8 matrix. Formula **596** in FIG. 5H define the off-diagonal blocks. These formulas for stabilization matrix are based on the initial element coordinate system. The stabilization matrix is a symmetric matrix.

**[0060]** With reference now to FIG. 6, a block diagram illustrates a computing device **600** in which the present invention may be implemented, and in which code or instructions implementing the processes of the present invention may be located. The exemplary computer system in FIG. 6 is discussed only for descriptive purposes. It should not be considered a limitation of the invention. Although the following descriptions related to a particular computer system, the concepts apply equally to other computer systems that are dissimilar to that shown in FIG. 6.

**[0061]** Computer system **600** includes at least one processor **610** and main random access memory (RAM) **620** connecting to a local bus **605** through a bridge **615**. Additional connections to local bus **605** may be made through direct component interconnection or through add-in boards. In the depicted example, network adapter **625**, small computer system interface (SCSI) adapter **630**, and expansion bus interface **635** are directly connected to local bus **605**. In contrast, audio adapter **640**, graphics adapter **645**, and video adapter **650** are connected to local bus **605** by add-in boards inserted into expansion slots.



Expansion bus interface **635** provides a connection for a keyboard and mouse adapter **655**, modem **660**, and additional memory **665**. SCSI adapter **630** provides a connection for hard disk drive **670**, tape drive **675**, and CD-ROM drive **680**.

**[0062]** In order to communicate with other computer systems via a network, the computer system **600** connects to the network via network adapter **625**. The network, Internet or intranet, connects multiple network devices utilizing general purpose communication lines.

**[0063]** Those of ordinary skill in the art will appreciate that the hardware shown in FIG. 6 may vary depending on the implementation. Other internal hardware or peripheral devices, such as flash ROM (or equivalent nonvolatile memory) or optical disk drives and the like, may be used in addition to or in lieu of the hardware depicted in FIG. 6. Also, the processes of the present invention may be applied to a multiprocessor computer system. In general, Computer system **600** is controlled and coordinated by operating system (OS) software, which performs tasks such as process scheduling, memory management, networking and I/O services. Exemplary OS includes Linux™, Microsoft Windows™.

**[0064]** Although an exemplary embodiment of invention has been disclosed, it will be apparent to those skilled in the art that various changes and modifications may be made to achieve the advantage of the invention. It will be obvious to those skilled in the art that some components may be substituted with another component providing same function. The appended claims cover the present invention.